

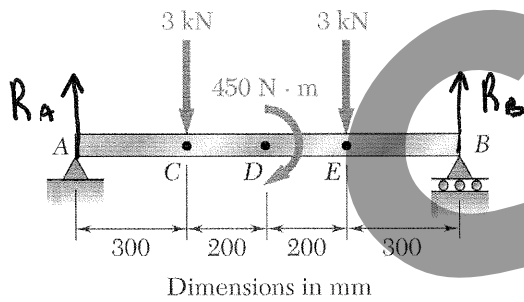
Name: **Solution**
 ID:
 Section:

Exam II

Monday May 25, 2009

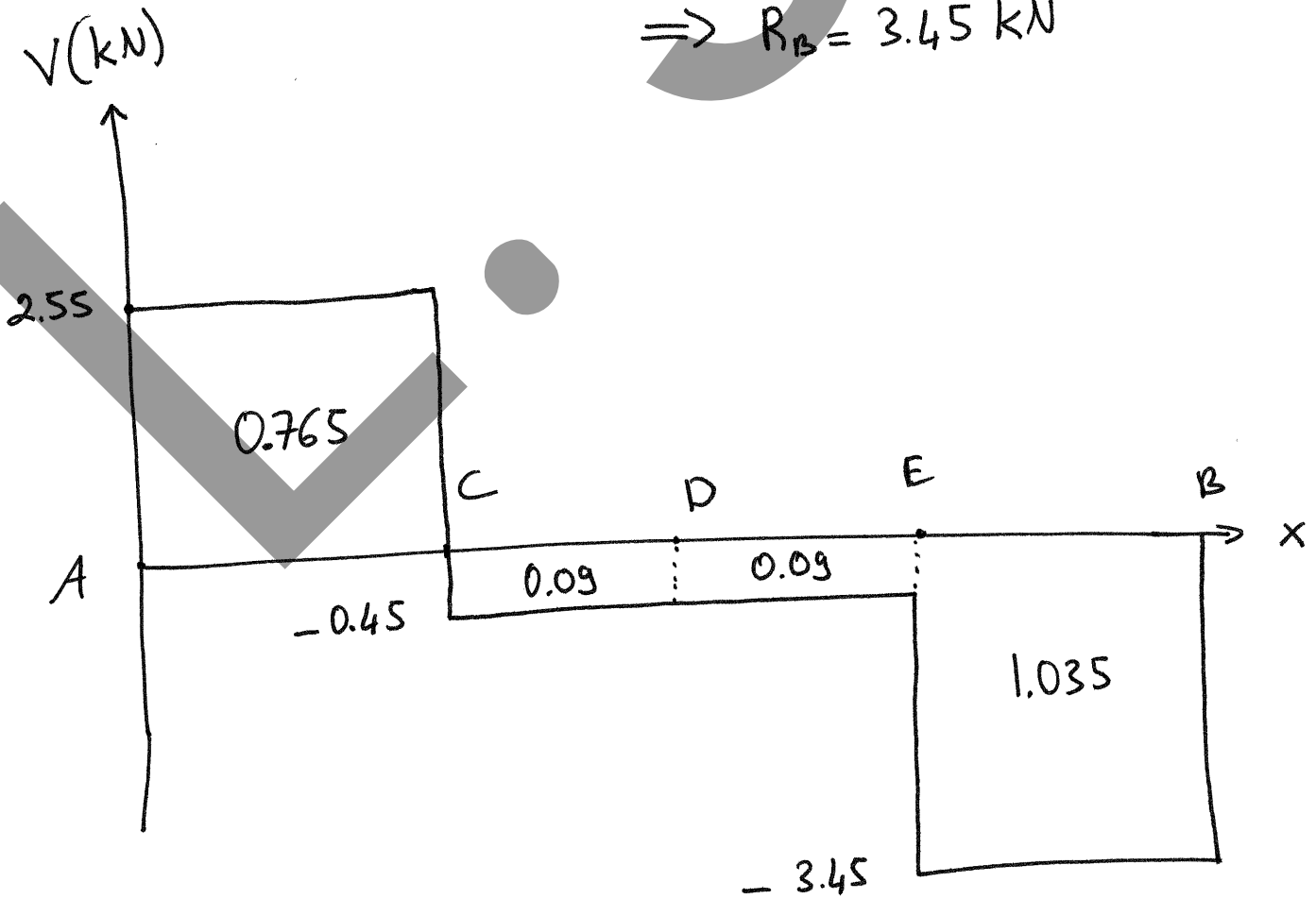
All work **must** be shown to receive full credit.

Problem 1. (25 pts.) Draw the shear and bending-moment diagrams for the beam and loading shown, and determine the maximum absolute value (a) of the shear, (b) of the bending moment.

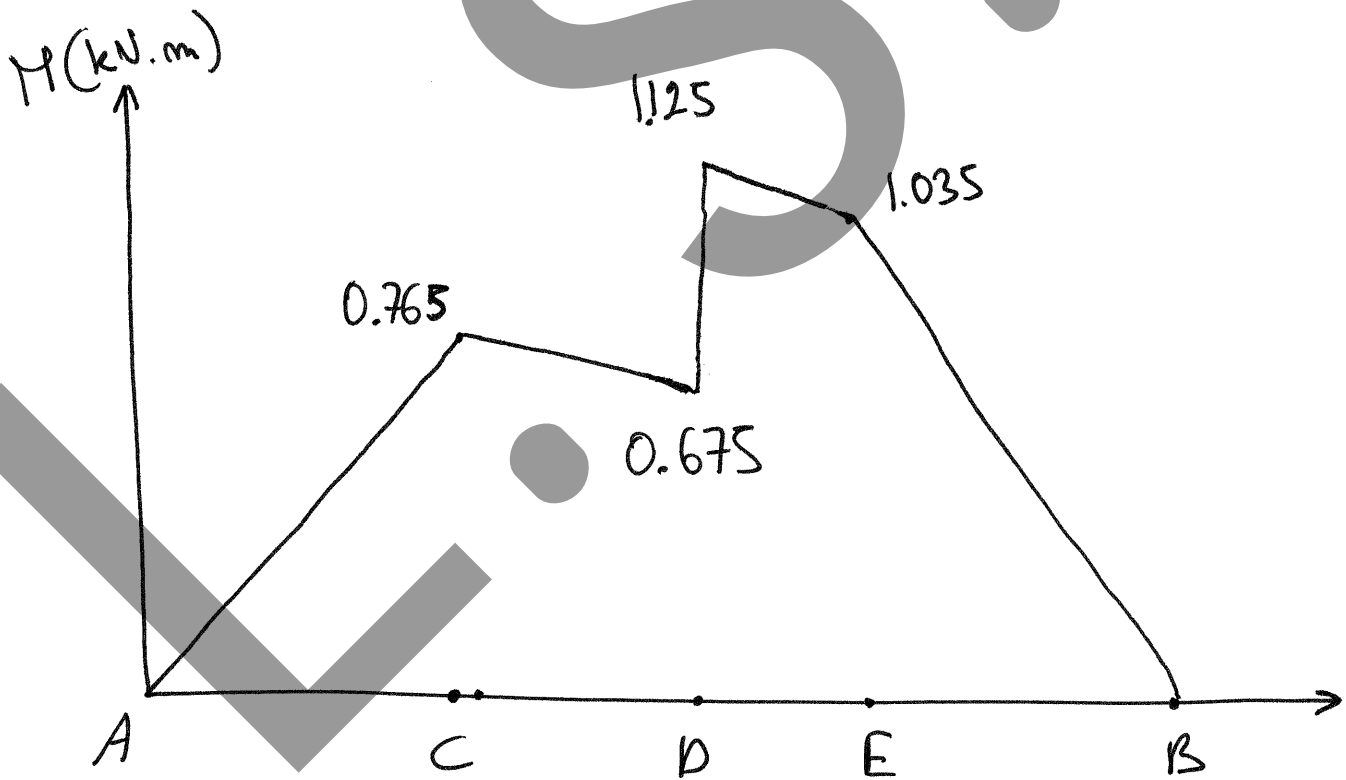


$$\begin{aligned} \bullet \sum M_B = 0 \\ 3000 \times 0.7 + 3000 \times 0.3 - 450 - R_A \times 1 = 0 \\ \Rightarrow R_A = 2.55 \text{ kN} \end{aligned}$$

$$\begin{aligned} \bullet \sum M_A = 0 \\ 3000 \times 0.3 + 3000 \times 0.7 + 450 + R_B \times 1 = 0 \\ \Rightarrow R_B = 3.45 \text{ kN} \end{aligned}$$

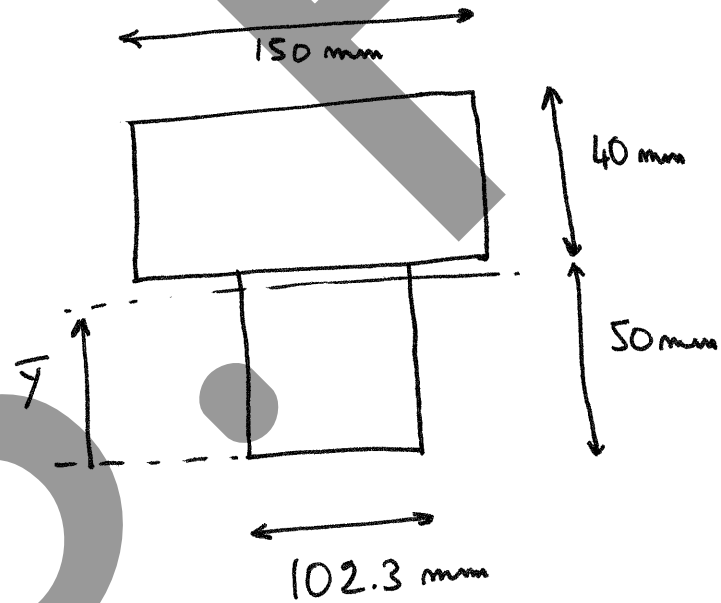
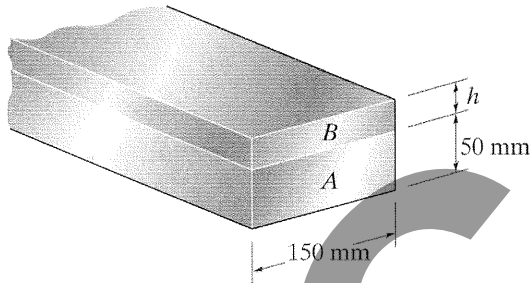


$$\begin{aligned}M_A &= 0 \\M_C &= M_A + 0.765 = 0.765 \\M_{D^-} &= M_C - 0.09 = 0.675 \\M_{D^+} &= M_{D^-} + 0.45 = 1.125 \\M_E &= M_{D^+} - 0.09 = 1.035 \\M_B &= 0\end{aligned}$$



$$\begin{aligned}\text{Maximum } |V| &= 3.45 \text{ kN} \\ \text{Maximum } |M| &= 1.125 \text{ kN.m}\end{aligned}$$

Problem 2. (25 pts.) The composite beam is made of Aluminum (A) and brass (B). If the height $h = 40 \text{ mm}$, determine the maximum moment that can be applied to the beam if the allowable bending stress for the aluminum is $(\sigma_{all})_A = 128 \text{ MPa}$ and for the brass $(\sigma_{all})_B = 35 \text{ MPa}$. ($E_A = 68.9 \text{ GPa}$ and $E_B = 101.0 \text{ GPa}$)



$$m = \frac{E_A}{E_B} = \frac{68.9}{101} = 0.682$$

$$b_A = m b_B = 0.1023 \text{ m}$$

$$\bar{y} = \frac{102.3 \times 50 \times 25 + 150 \times 40 \times 70}{102.3 \times 50 + 150 \times 40} = 49.3 \text{ mm}$$

$$I = I_A + I_B$$

$$= \frac{1}{12} 102.3 \times 50^3 + 102.3 \times 50 \times 24.3^2 + \frac{1}{12} 150 \times 40^3 + 150 \times 40 \times 20.7^2$$

$$I = 7.45 \times 10^{-6} \text{ m}^4$$

• Failure in brass : B

$$\sigma_{allB} = \frac{M_b C_b}{I} \quad C_b = 0.0407 \text{ m}$$

$$\Rightarrow M_B = \frac{35 \times 10^5 \times 7.45 \times 10^{-6}}{0.0407} = 6.4 \text{ kN.m}$$

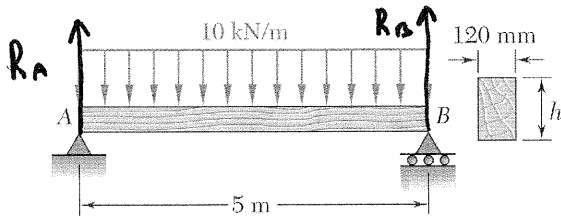
• Failure in Aluminum: A

$$\sigma_{allA} = \frac{M_A C_A}{I} \Rightarrow C_A = 0.0493 \text{ m}$$

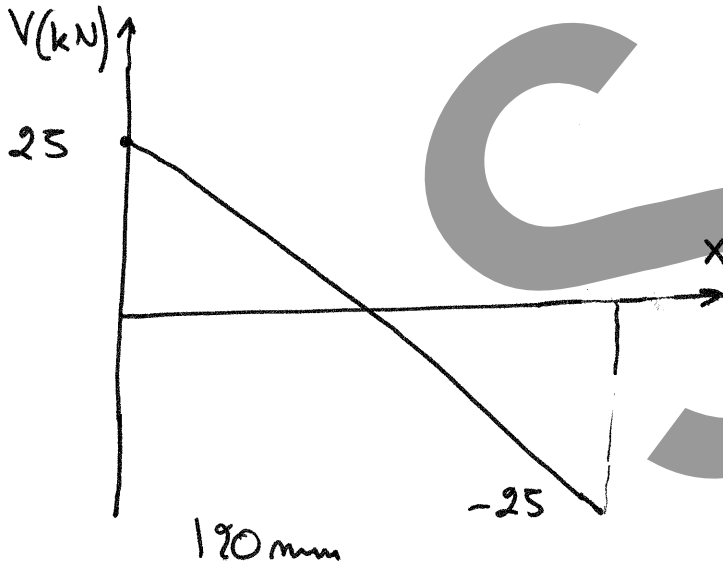
$$\Rightarrow M_A = \frac{128 \times 10^6 \times 7.45 \times 10^{-6}}{0.682 \times 0.0493} = 28.3 \text{ kN.m}$$

$$M_{\max} = 6.4 \text{ kN.m}$$

Problem 3. (25 pts.) The timber beam is designed to support the distributed loading shown. Knowing that for the grade of timber used $\tau_{all} = 0.8 \text{ MPa}$, determine the minimum required depth h of the beam.

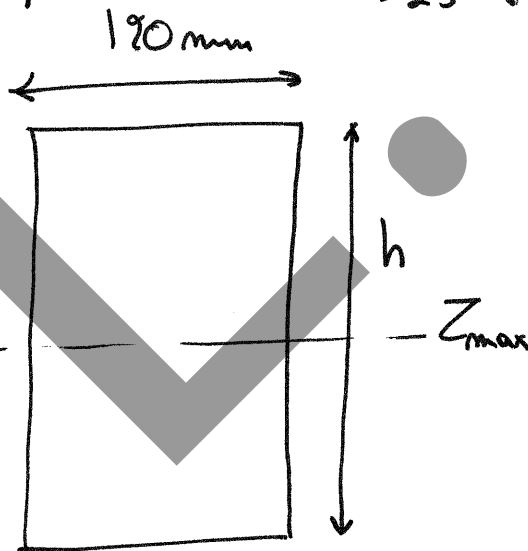


$$R_A = R_B = 25 \text{ kN}$$



Maximum V occurs at A or B

$$|V_{max}| = 25 \text{ kN}$$



$$\tau_{max} = \frac{VQ}{It} = \tau_{all}$$

$$Q = \frac{h}{2} \times 0.12 \times \frac{h}{4} = 0.015h^2$$

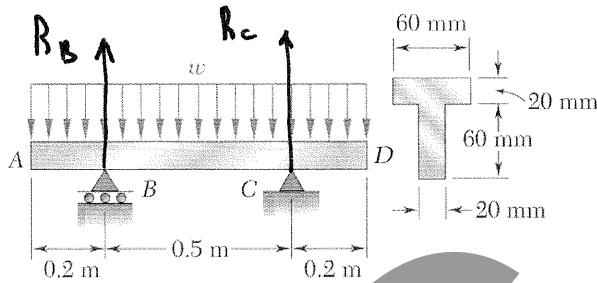
$$t = 0.12$$

$$I = \frac{1}{12} 0.12 \times h^3$$

$$\Rightarrow 0.8 \times 10^6 = \frac{25 \times 10^3 \times 0.015h^2}{\frac{1}{12} \times 0.12^2 \times h^3} \Rightarrow h = 0.3906$$

The minimum required depth h is $h_{mi} = 390.6 \text{ mm}$

Problem 4. (25 pts.) Determine the largest permissible distributed load w for the beam shown, knowing that the allowable normal stress is $+84 \text{ MPa}$ in tension and -205 MPa in compression.



$$R_B = R_C = 0.45w$$

$$V_A = 0$$

$$V_B^- = V_A - 0.2w = -0.2w$$

$$V_B^+ = V_B^- + R_B = 0.25w$$

$$V_C^- = V_B^+ - 0.5w = -0.25w$$

$$V_C^+ = V_C^- + R_C = 0.2w$$

$$V_D = 0$$

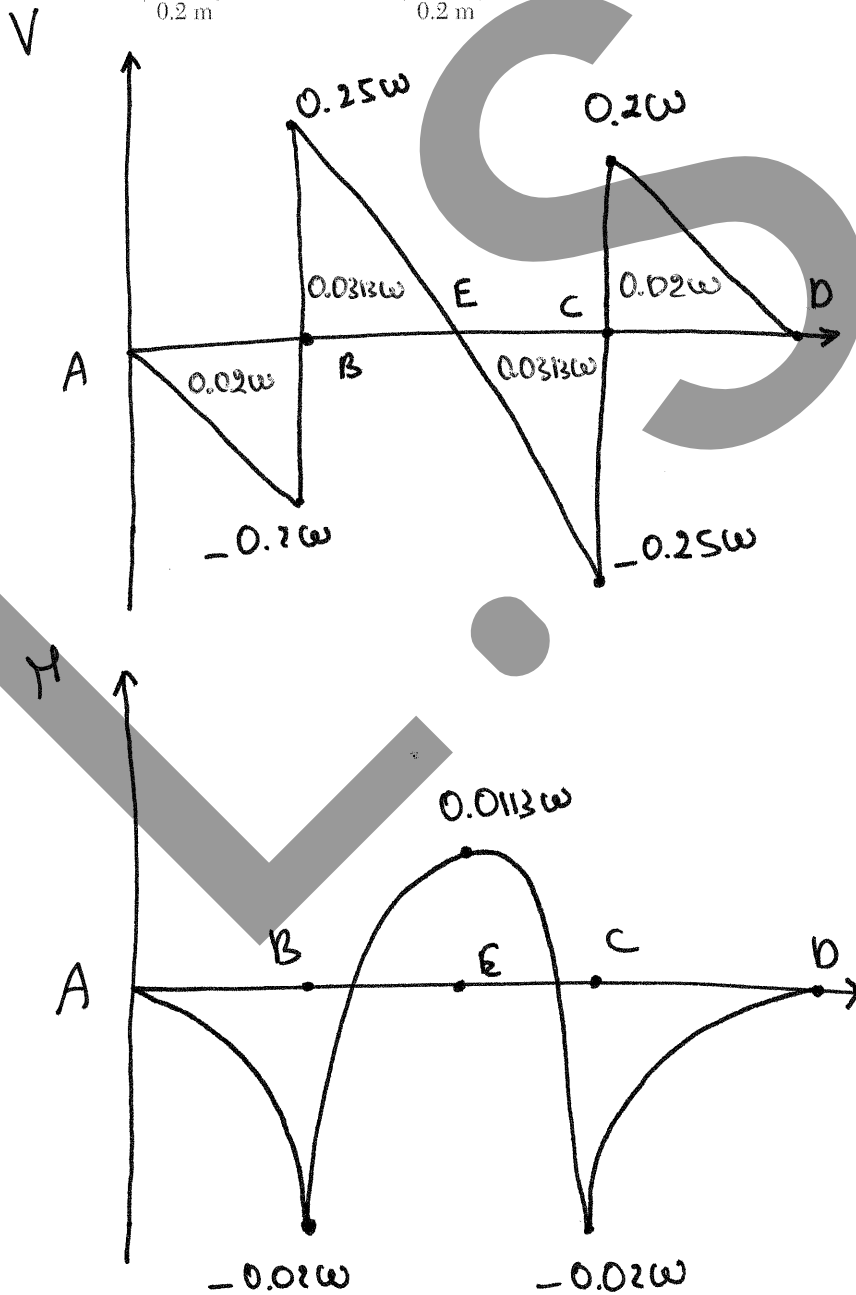
$$M_A = 0$$

$$M_B = M_A - 0.02w = -0.02w$$

$$M_E = M_B + 0.0313w = 0.0113w$$

$$M_C = M_E - 0.0313w = -0.02w$$

$$M_D = 0$$



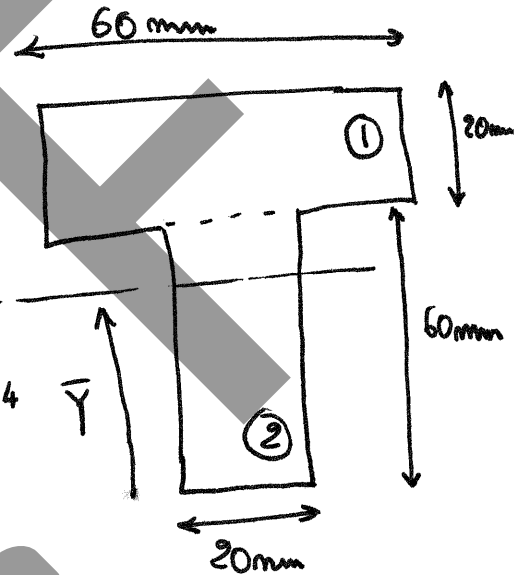
$$\bar{Y} = \frac{60 \times 20 \times 30 + 60 \times 20 \times 70}{60 \times 20 + 60 \times 20} = 50 \text{ mm}$$

$$I = I_1 + I_2$$

$$= \frac{1}{12} 20 \times 60^3 + 20 \times 60 \times 20^2$$

$$+ \frac{1}{12} 60 \times 20^3 + 60 \times 20 \times 20^2 = 1360 \times 10^3 \text{ mm}^4$$

N.A



$$C_{\text{top}} = 30 \text{ mm}$$

$$C_{\text{bottom}} = 50 \text{ mm}$$

• At B and c: $M = -0.02w$

Tension at top $V_{all} = \frac{M c_{\text{top}}}{I} \Rightarrow M = \frac{V_{all} I}{C_{\text{top}}} = 0.02w$

$$\Rightarrow w = \frac{84 \times 10^6 \times 1360 \times 10^{-9}}{0.02 \times 0.03} = 190.4 \times 10^3 \text{ N/m}$$

Compression at bottom $\Rightarrow w = \frac{V_{all} c}{I} = \frac{205 \times 10^6 \times 1360 \times 10^{-9}}{0.02 \times C_{\text{bottom}}}$

$$\Rightarrow w = 278.8 \times 10^3 \text{ N/m}$$

• At E: $M = 0.0113w$

Tension at bottom $\Rightarrow w = \frac{V_{all} I}{0.02 C_{\text{bottom}}} = \frac{84 \times 10^6 \times 1360 \times 10^{-9}}{0.0113 \times 0.05} = 202 \times 10^3 \text{ N/m}$

Compression at Top $\Rightarrow w = \frac{V_{all} c}{0.02 \times C_{\text{top}}} = \frac{205 \times 10^6 \times 1360 \times 10^{-9}}{0.0113 \times 0.03} = 826 \times 10^3 \text{ N/m}$

$$\Rightarrow \boxed{w_{\text{max}} = 190.4 \text{ kN/m}}$$